

# SOLAR THERMAL ENERGY DRIVEN ORGANIC RANKINE CYCLE SYSTEMS FOR ELECTRICITY AND FRESH WATER GENERATION

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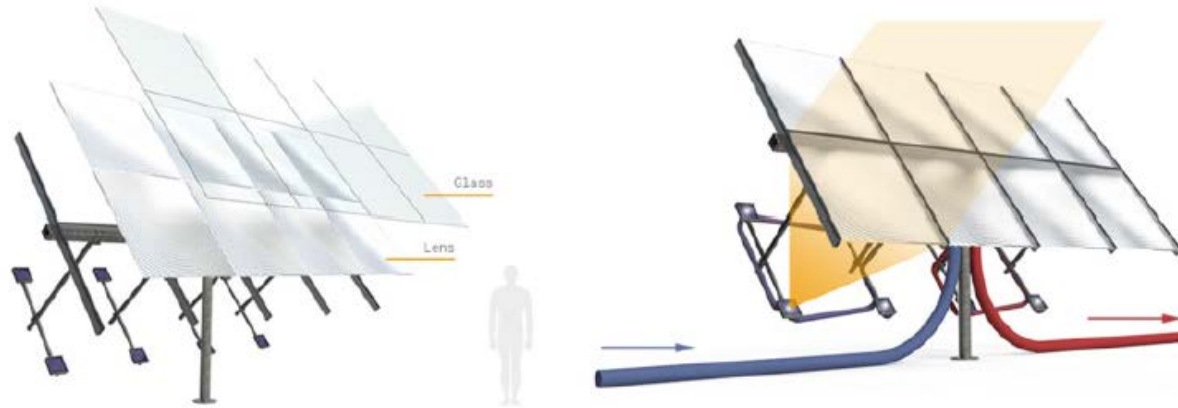
# Introduction

- **Small to medium-scale systems for dispatchable electricity and fresh water**
  - Diesel generator based systems using reverse osmosis (RO)
  - Solar photovoltaic with battery storage system using reverse osmosis
  - Expensive large-scale battery systems
  - Concentrated solar power (CSP)
    - **Thermal energy storage**
    - **Thermal energy driven desalination system**

# Conventional CSP plants

- **Parabolic trough collector (PTC), linear Fresnel reflector (LFR), solar tower, paraboloid dish**
- **Heavy and very expensive glass mirrors and receivers**
- **High capital and O&M cost**
- **Suitable for large-scale and places with high solar radiation**

# Foil-based concentrating solar collector



*Actual installation on Møn, Denmark (a 2 MW solar field for district heating purposes) and at DTU Civil Engineering's test facilities*

# Comparison between foil-based solar field and parabolic trough collector field

	Foil-based solar field	Parabolic trough collector field
1	Low cost concentrators	High cost concentrators
2	Low precision requirements	High precision requirements
3	Standard components	Specialized components
4	Light construction	Heavy construction
5	Two-axis tracking	Single-axis tracking
6	Less sensitive to dirt	Difficult to clean
7	Any terrain usable	Level terrain required

- **Potential to reduce about 40 % LCOE compared to medium-scale PTC-based plants [1]**

## CSP integrated ORC system with reverse osmosis desalination

- Promising working fluids: Toluene, isopentane, R134a, hexamethyldisiloxane (MM) and n-octane

## CSP integrated ORC system with thermal desalination

- Parabolic trough collector preferred solar field
- Promising working fluid: Toluene, cyclopentane and n-pentane

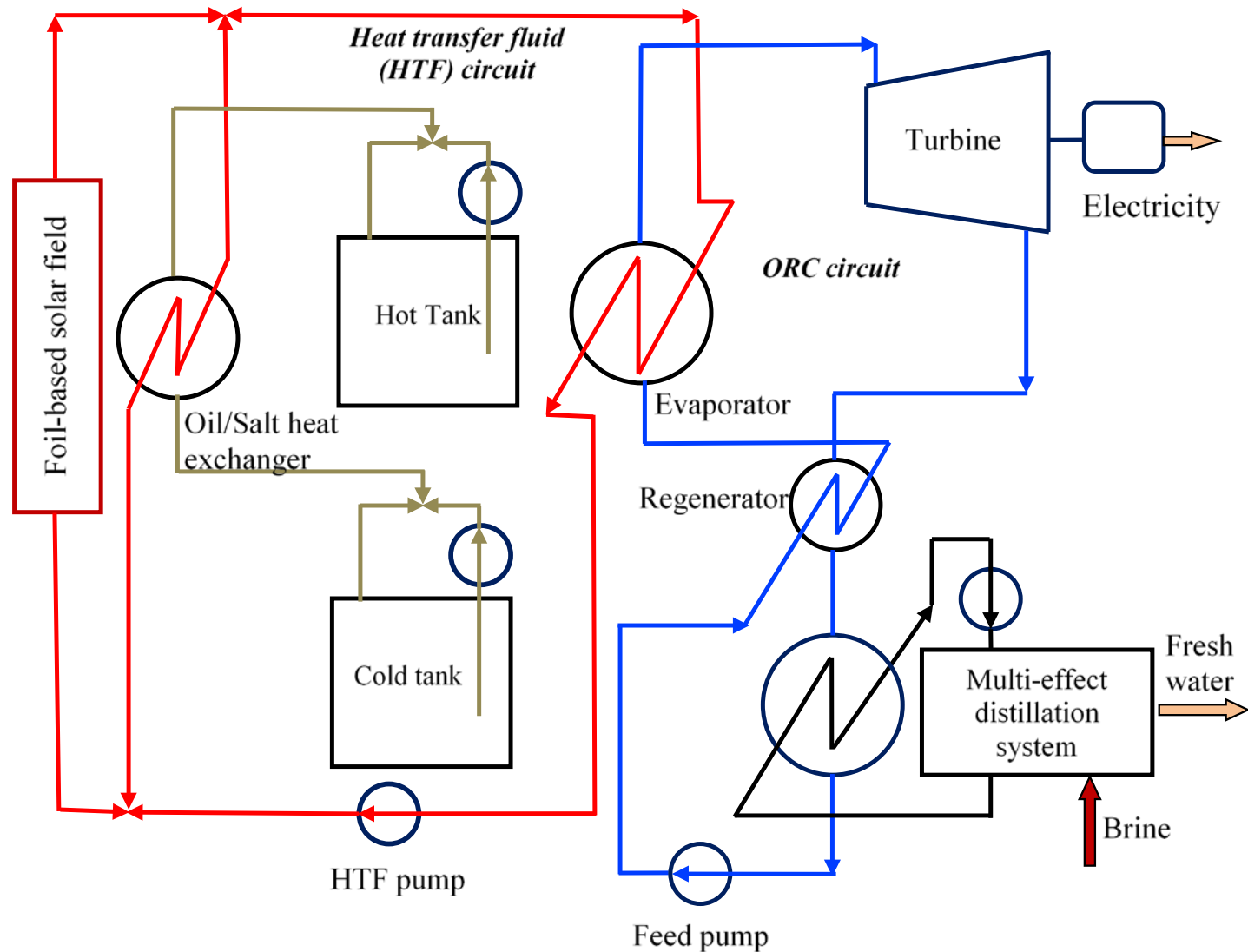
# List of concentrated solar energy powered organic Rankine cycle based commercial/medium-scale actual plants for different applications



Name (Location)	Start year	Solar field	Solar field area (m <sup>2</sup> )	Storage	Application (net capacity)
<b>Saguaro Power Plant (Arizona, USA)</b>	2006	PTC	10,340	-	Electricity generation (1 MW <sub>e</sub> ) (currently non-operational).
<b>Rende-CSP Plant (Calabria, Italy)</b>	2014	LFR	9,780	-	Electricity generation (1 MW <sub>e</sub> ). The facility is combined with an already operating biomass based plant (14 MW <sub>e</sub> ).
<b>Airlight Energy Ait-Baha Pilot Plant (Ait Baha, Morocco)</b>	2014	PTC	6,159	Packed-bed rock (5 h)	Electricity generation from CSP and waste heat from cement industry (hybrid plant) (2 MW <sub>e</sub> ).
<b>Stillwater GeoSolar Hybrid Plant (Fallon, USA)</b>	2015	PTC	24,778	-	Electricity generation. About 17 MW <sub>th</sub> from CSP combined with geothermal energy producing 33 MW <sub>e</sub> . Additionally, 26.4 MW <sub>e</sub> of a solar photovoltaic plant.
<b>Aalborg CSP-Brønderslev CSP with ORC project (Brønderslev, Denmark)</b>	2016	PTC	26,929	-	Combined heat and electricity production from CSP (16.6 MW <sub>th</sub> ) and biomass combustion (hybrid plant) (3.8 MW <sub>e</sub> ).
<b>Ottana Solar Facility (Sardinia, Italy)</b>	2017	LFR	8,600	Two-tank direct	Power generation (0.6 MW <sub>e</sub> ), additionally 0.4 MW <sub>e</sub> of solar PV



# Foil-based solar collector powered system



# Important data used for the analysis

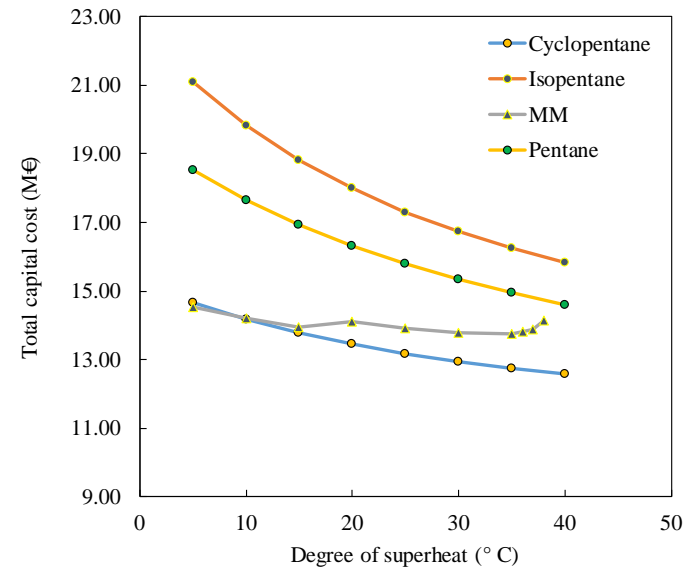
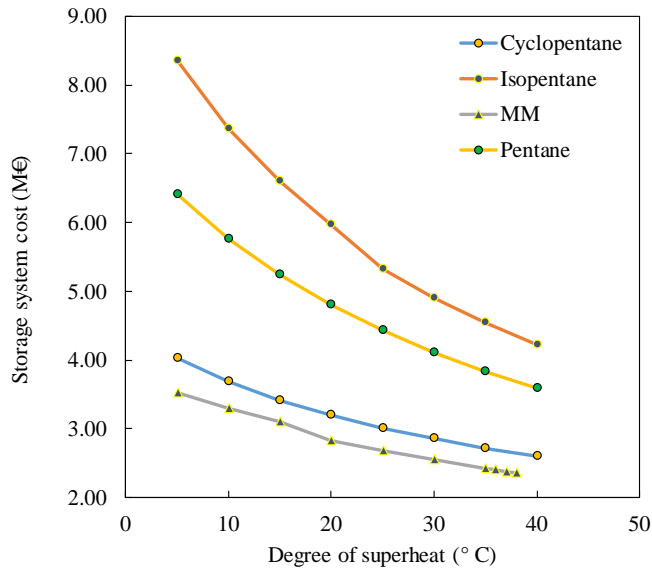
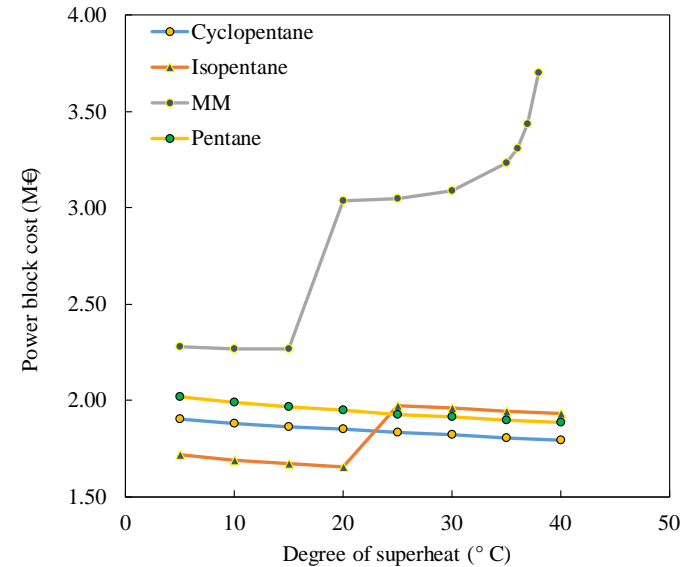
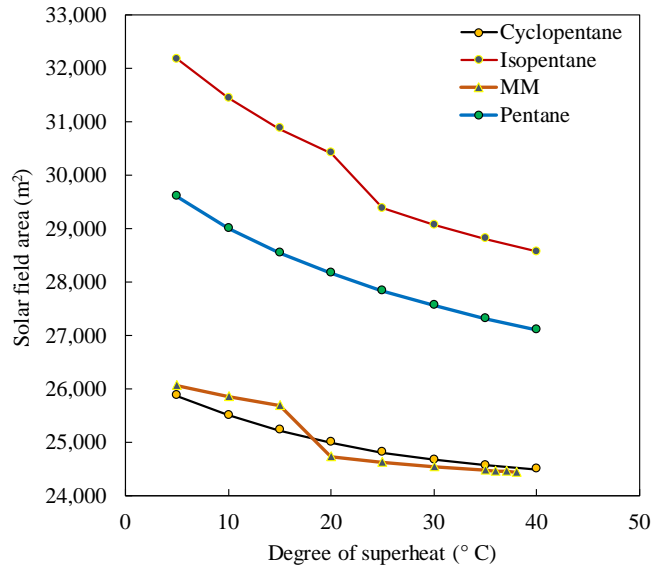
Input Parameter	Value
Place	Cape Town (South Africa)
Solar collector field efficiency parameters	$\eta_{o,CL} = 0.833$ ; $U_1 = 0.85 \text{ W}/(\text{m}^2 \cdot \text{K})$ and $0.68 \text{ W}/(\text{m}^2 \cdot \text{K})$ (with improvements)
Storage tank UA	$0.00017 \cdot T_{\text{salt}} + 0.012 \text{ (kW/m}^2)$ ; $T_{\text{salt}}$ is the temperature of salt ( $^{\circ} \text{C}$ ) [2]
Gross peak plant output at design ( $P_{\text{gross,D}}$ )	1 MWe
Penalty for start-up/shut-down and other losses	10 % of net power output [3]
Parasitic electric energy use for solar field, TES pump, antifreeze pumping	7 % of net power output [4]
Electricity consumption of MED system	$1 \text{ kWh}_e/\text{m}^3$ [5]
Isentropic efficiency of turbine at design condition ( $\eta_{T,D}$ )	Based on [6]
Turn down ratio of turbine ( $P_{\text{min}}/P_{\text{max}}$ )	0.1 [7]
Generator efficiency parameters ( $\eta_{g,D}$ )	$\eta_{g,D} = 0.93$
Isentropic efficiency of pump at design condition ( $\eta_{p,D}$ )	$\eta_{p,D} = 0.7$

# Important data used for the economic analysis



Parameters	Value
Solar field and heat transfer fluid system cost (€)	250 (for current plants) and 150 (for future plants)
Land and site development cost (€/m <sup>2</sup> of land)	4.7 [8]
Specific cost of Hitec XL (€/kg)	0.93 [9]
Storage tank cost (including insulation and foundation) (€/m <sup>3</sup> )	$\frac{C_{ST,Tank}}{C_{ref,Tank}} = \left( \frac{Capacity_{ST,Tank}}{Capacity_{ref,Tank}} \right)^{0.85}$ ; $C_{ref,Tank} = 7,932,160$ € [1], $Capacity_{ref,Tank} = 30,844 \text{ m}^3$ [2]
Oil to salt heat exchanger cost (€)	$\frac{C_{ST,HX}}{C_{ref,HX}} = \left( \frac{Capacity_{ST,HX}}{Capacity_{ref,HX}} \right)^{0.85}$ ; $C_{ref,HX} = 4,580,235$ € [1], $Capacity_{ref,HX} = 133 \text{ MW}_{th}$ [2]
Balance of storage system and salt pump cost (€)	14 % of the storage system cost [2]
ORC system component cost (€)	Based on the correlations given by [10] and water cooled condenser based on [11]
Capital cost of MED system (€/(m <sup>3</sup> ·d))	1500
Annual operating and maintenance cost	Fixed cost (on total investment) – 1.4 %; Variable cost – 2.8 €/MWh <sub>e</sub>
Lifetime (y)	25

# Variations in degree of superheat



# Results of a techno-economic analysis of the system (1 MW<sub>e</sub> plant with 9 h storage capacity)



Parameters	Cyclopentane		MM		N-Pentane		Isopentane	
	Current plants	Future plants	Current plants	Future plants	Current plants	Future plants	Current plants	Future plants
Solar field outlet temperature, $T_{out,CL}$ (° C)	278		290		256		247	
Evaporator pressure, $P_{eva}$ (MPa)	2.569		1.201		2.467		2.464	
Evaporator outlet temperature, $T_{out,eva}$ (° C)	238		250		216		207	
Solar collector field area, $A_{p,CL}$ (m <sup>2</sup> )	24,498	21,418	24,476	21,148	27,098	24,004	28,567	25,423
MED system capacity (m <sup>3</sup> /day)	908		862		1,087		1,180	
LCOE (€/kWh <sub>e</sub> )	0.224	0.176	0.277	0.226	0.238	0.188	0.255	0.206
LCOW (€/m <sup>3</sup> )	1.74	1.27	1.48	1.02	2.12	1.65	2.24	1.75

# Concluding remarks

- **Foil-based CSP system:** Promising option
- **ORC working fluid:** Cyclopentane
  - **LCOE:** 5.6–6.1 % lower compared to n-pentane
  - **LCOE:** 19.6–22.7 % lower compared to MM based plants
  - **Levelized cost of water (LCOW):** 19.8–39.6 % higher compared to MM
  - **LCOW:** 25.5–32.3 % lower compared to n-pentane based plants

# References

- [1] Desai, N.B., Pranov, H., Haglind, F., 2019, Techno-economic analysis of a power generation system consisting of a foil-based concentrating solar collector and an organic Rankine cycle unit. In Proceedings of the 32nd International conference on efficiency, cost, optimization, simulation and environmental impact of energy systems (ECOS 2019).
- [2] Herrmann, U., Kelly, B., Price, H., 2004, Two-tank molten salt storage for parabolic trough solar power plants. *Energy*, vol. 29(5-6): pp. 883–893.
- [3] Manzolini, G., Bellarmino, M., Macchi, E., Silva, P., Solar thermodynamic plants for cogenerative industrial applications in southern Europe. *Renew Energy* 2011;36(1):235–243.
- [4] NREL, 2018. National Renewable Energy Laboratory, <[www.sam.nrel.gov/](http://www.sam.nrel.gov/)> accessed 15.12.2018.
- [5] Alfa Laval, 2018, <[www.alfalaval.com/globalassets/documents/products/process-solutions/desalination-solutions/multi-effect-desalination/fresh-water-brochure-pee00251en-1201.pdf](http://www.alfalaval.com/globalassets/documents/products/process-solutions/desalination-solutions/multi-effect-desalination/fresh-water-brochure-pee00251en-1201.pdf)>, accessed 01.07.18.

# References (cont.)

- [6] Astolfi, M., Macchi, E., 2015, Efficiency correlations for axial flow turbines working with non-conventional fluids. In Proceedings of the 3rd International Seminar on ORC Power Systems, Brussels, Belgium: pp. 12-14.
- [7] Turboden, 2018. <[www.turboden.com](http://www.turboden.com)> accessed 22.11.2018.
- [8] IIT Bombay, Solar Thermal Simulator Version 2.0, 2014.
- [9] Pan, C.A., Ferruzza, D., Guédez, R., Dinter, F., Laumert, B., Haglind, F., Identification of optimum molten salts for use as heat transfer fluids in parabolic trough CSP plants: A techno-economic comparative optimization. In AIP Conference Proceedings 2018;2033(1):030012.
- [10] Astolfi, M., Romano, M.C., Bombarda, P., Macchi, E., 2014, Binary ORC (Organic Rankine Cycles) power plants for the exploitation of medium–low temperature geothermal sources– Part B: Techno-economic optimization, Energy, 66: pp. 435-446.
- [11] Lemmens, S., 2016, Cost engineering techniques and their applicability for cost estimation of organic Rankine cycle systems. Energies, vol. 9(7): pp. 485.



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# Thank you for your attention